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$$\triangle = \begin{vmatrix} -\frac{yz}{x^2}, & \frac{z}{x}, & \frac{y}{x} \\ \frac{z}{y}, & -\frac{xz}{y^2}, & \frac{x}{y} \\ \frac{y}{z}, & \frac{x}{z}, & -\frac{xy}{z^2} \end{vmatrix} = xyz \begin{vmatrix} -\frac{1}{x}, & \frac{1}{y}, & \frac{1}{z} \\ \frac{1}{x}, & -\frac{1}{y}, & \frac{1}{z} \\ \frac{1}{x}, & \frac{1}{y}, & -\frac{1}{z} \end{vmatrix} = xyz \begin{vmatrix} 0, & 0, & \frac{2}{z} \\ \frac{2}{x}, & 0, & 0 \\ \frac{1}{x}, & \frac{1}{y}, & -\frac{1}{z} \end{vmatrix} = 4.$$

The value announced, 4xyz, is therefore erroneous.

Solved similarly by J. Scheffer and W. J. Greenstreet.

## MECHANICS.

## 356. Proposed by the late G. B. M. ZERR, Ph. D.

A cantilever beam length a is loaded with c pounds per running foot at its fixed end and increases uniformly to b pounds per running foot at its free end. Find the deflection at the free end due to this load.

## Solution by FRANCIS RUST, E. E., Pittsburg, Pa.

The problem is solved by determining the elastic curve of the beam's axis from its second differential equation

$$y'' = \frac{M}{IE}$$

as derived from Hooke's law, to be found in all text books. M is the bending moment in the point, abscissa=x; I is the moment of inertia of the beam's cross-section, and E is the modulus of elasticity of its material.

M is to be determined by the arrangement of the load with mathematical certainty. Taking the point, abscissa=x, for origin,

$$M = \int_{0}^{a-x} q^{\xi} d\xi,$$

q, the load per lineal foot, to be expressed as a function of the variable of integration  $\xi$ . q=c+px, and c+pa=b. Consequently,

$$p = \frac{b-c}{a}$$

and q as a function of  $\xi$  is

$$q = c + \frac{b - c}{a}$$
.  $(x + \xi) = \frac{ac + (b - c)x}{a} + \frac{b - c}{a}\xi$ , and

$$\begin{split} M = & \frac{ac + (b - c)x}{a} \int_{0}^{a - x} \xi \, d \, \xi + \frac{b - c}{a} \int_{0}^{a - x} \xi^{2} d \, \xi = \frac{ac + (b - c)x}{a} \cdot \frac{(a - x)^{2}}{2} \\ & + \frac{b - c}{a} \cdot \frac{(a - x)^{3}}{3} = \frac{a^{2}}{6} \left(1 - \frac{x}{a}\right)^{2} \left((2b + c) + (b - c)\frac{x}{a}\right) \end{split}$$

which, properly simplified, gives us

$$M = \frac{\alpha^2}{6} \left[ (2b+c) - 3(b+c) \frac{x}{a} + 3c(\frac{x}{a})^2 + (b-c)(\frac{x}{a})^3 \right].$$

Assuming now a beam of uniform cross-section and homogeneous material, and introducing the new variable, x/a=u, we have dx=adu.

The integration of y'' = M/IE may be performed without any difficulty. It gives the equation of the elastic curve,

$$y = \frac{a^4}{6EI} \cdot \left[ \frac{2b+c}{2} \left( \frac{x}{a} \right)^2 - \frac{b+c}{2} \left( \frac{x}{a} \right)^3 + \frac{c}{4} \left( \frac{x}{a} \right)^4 + \frac{b+c}{20} \left( \frac{x}{a} \right)^5 \right]$$

from which is derived for x=a,

$$y_a = f = \frac{a^4 (11b + 4c)}{120EI}$$
.

Remark. This result is as simple as it is wrong.

In my memoir *Der Fehler in Hooke's Gesetz*, published in Oesten, Wochensch. f. d. aff. Bandienst, year 1900, p. 252, ff., I have deduced mathematically and proved by experiment, that Hooke's law *cannot be applied on the beginning of deformation*.

The locus of any elastic curve, deduced from Hooke's law, p. e. our equation above, is a continuous curve, whilst the experiment shows, that a cantilever (spring under its own weight, for instance) will remain unbent for a certain length at its free end.

The investigation into these conditions reveals the fact, that any material may be submitted to a certain stress, which must be overcome before deformation takes place. Such minimal stress may be called *the resistance* of inertia (Tragheits widerstand). Denoting it by  $\alpha$ , the true expression of the law of elastic deformation is

$$\triangle l: l = \sqrt{(\sigma^2 + a^2)} : E.$$